



# ***Z-Pinch Power Plant Shock Mitigation Experiments, Modeling, and Code Assessment***

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# ***Introduction***

- **Problem Statement**
- **Aluminum Foam Experiments**
- **Analytic Models**
- **Aluminum Foam Simulations**
- **Conclusion**





# ***Problem Statement***

- **Mitigate the powerful shock wave that is generated inside the Z Pinch Power Plant.**
- **Fusion of deuterium-tritium pellets:**  
$${}^2_1H + {}^3_1H \rightarrow {}^4_2He + {}^1_0n + E$$
- **Each D-T capsule will release about 3 gigajoules (and perhaps more...).**
- **This is the equivalent energy released by a 1,500-lb bomb...**
- **...Or 1,911 jelly doughnuts.**
- **As a first step, we will conduct shock experiments to**
  - **Obtain data with which to benchmark structural analysis codes.**
  - **Explore the feasibility of metal foams as a shock mitigation medium in IFE chambers.**
  - **Validate EOS and constitutive equations.**





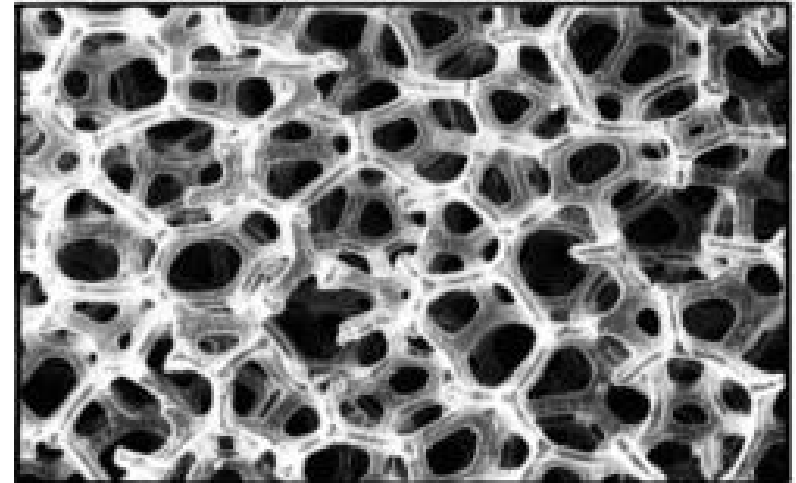
# ***Metal Foam Experiments***

- **The University of Wisconsin has already done shock experiments on thin liquid-water films.**
- **Metal foam appears promising in terms of shock mitigation. Consequently, we conducted a series of shock tests at the Shock Tube Facility in the UW.**
- **A variety of porosities, materials, test configurations, and shock wave magnitudes are being considered.**
- **20.3-cm thick samples will be tested in about three weeks. Because they are thicker, they will be subjected to higher shocks.**
- **A foamed thick-liquid water experiment will also be conducted.**



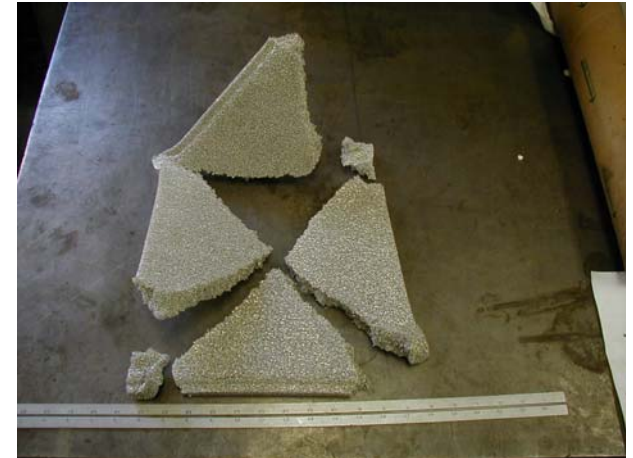
# ***Metal Foam Shock Mitigation Tests***

- **Purpose**
  - Provide data to benchmark codes
  - Explore feasibility as a shock mitigation medium
- **Test specimens**
  - 6101-T6 Aluminum
  - 10, 20, 40 ppi
  - 2.54 cm thick
- **Performed at WiSTL**
  - Single and dual layers



# Metal Foam Shock Mitigation Tests

- **Single layer**
  - **1.45 Ma – Sample failure**
  - **1.34 Ma**
    - Reflections subsonic
    - Transmitted shocks : 1.19-1.20 Ma
    - Deformation
      - ♦ 20, 40 ppi 31% - 38%
      - ♦ 10 ppi 8%
- **Double layer – 1.34 Ma**
  - **Transmitted shocks: 1.14, 1.16 Ma**
  - **Deformation**
    - Top: 21% & 25%
    - Lower: 24% & 48%
  - **48% deformation**
    - Off center
    - 1" separation
    - Intra-layer reflections?



Central  
Vertical  
Deformation



# ***Metal Foam Shock Mitigation Tests***

<b>Test</b>	<b>Sample No.</b>	<b>Porosity (ppi)</b>	<b>Configuration</b>	<b>Trans. Shock (Ma)</b>	<b>Central Deformation (cm / %)</b>
4	3	40	Single layer (Ma 1.45)	No data	Material failed
6	5	40	Single layer	1.19	0.961 / 38
7	10	40	Single layer	1.20	0.777 / 31
8	6 & 7	40	2 layer	1.14	0.538 / 21
			3" separation		0.593 / 24
9	8 & 9	40	2 layer	1.16	0.618 / 25
			1" separation		1.20 / 48*
10	2	20	Single layer	1.20	0.840 / 33
11	1	10	Single layer	1.19	0.212 / 8

\* Off-center deformation in lower layer.





## ***Code Selection and Analytic Models***

- We will use ALEGRA, ABACUS, BUCKY, CTH, and DYNA2D to simulate Shock Tube Test 6.
- We will investigate the following models, and perhaps others:
  - Mie-Grüneisen US UP (EOS)
  - Elastic Plastic Power Law Hardening (constitutive equation)
  - P- $\alpha$  (constitutive equation appropriate for void fractions less than 20%)





# Analytic Models

- **Mie-Grüneison (MG) US UP EOS Model:**

- **Pressure**

$$P(\rho, E) = P_H(\rho) + \Gamma_0 \rho_i [E - E_H(\rho)]$$

- **Energy**

$$E(\rho, T) = E_H(\rho) + C_v [T - T_H(\rho)]$$

- **Shock and Particle Velocity**

$$U_s = C_0 + S_1 u_p$$

( $P_H$  = Hugoniot pressure,  $E_H$  = Hugoniot energy,  $T_H$  = Hugoniot temperature,  $T$  = material temperature,  $E$  = material energy,  $P$  = material pressure,  $\rho_i$  = initial density,  $\rho$  = density,  $\Gamma_0$  = Grüneisen parameter,  $C_v$  = specific heat,  $U_s$  = shock velocity,  $c_0$  = bulk sound speed,  $S_1$  = slope of the U-u Hugoniot, and  $u_p$  = particle velocity.)

- In ALEGRA, the MG model can be easily modified to incorporate the P- $\alpha$  model.



# Analytic Models

$$\text{From } C_L = \sqrt{\frac{B + \frac{4}{3}G}{\rho}},$$

$$C_S = \sqrt{\frac{G}{\rho}},$$

$$B = \frac{E}{3(1-2\nu)},$$

$$G = \frac{E}{2(1+\nu)}, \text{ and}$$

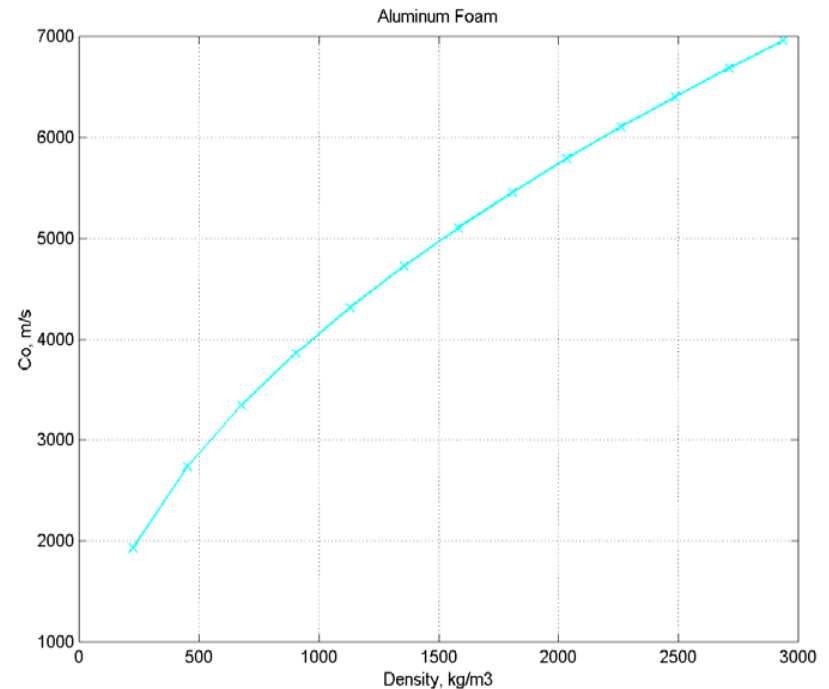
$$E = AE_0 \left( \frac{\rho}{\rho_0} \right)^2 \text{ (Gibson - Ashby Data Fit)}$$

$$\Rightarrow C_L = \frac{1}{\rho_0} \left[ \frac{\rho AE_0 (1-\nu)}{(1-2\nu)(1+\nu)} \right]^{\frac{1}{2}} \text{ and}$$

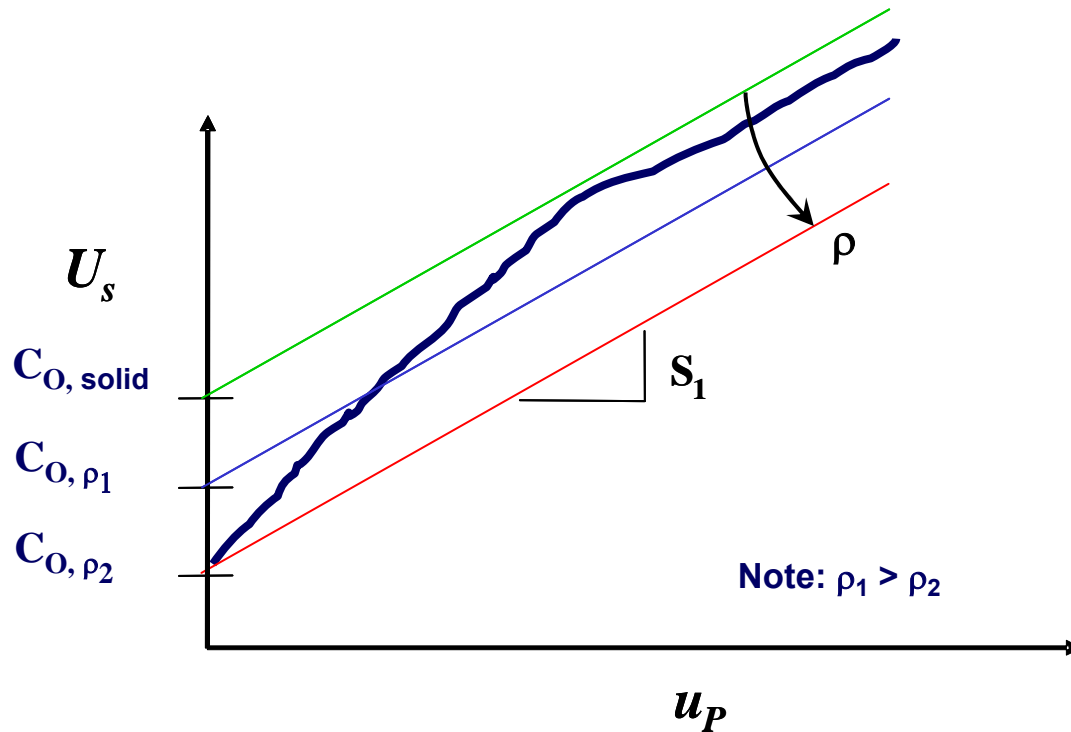
$$\Rightarrow C_S = \frac{1}{\rho_0} \left[ \frac{\rho AE_0}{2(1+\nu)} \right]^{\frac{1}{2}}.$$

Approx:  $C_o \approx \sqrt{C_L^2 + C_S^2}$  for mild shocks

$C_L$  = compressive wave velocity,  $C_S$  = shear wave velocity,  
 $C_o$  = bulk sound speed,  $B$  = bulk modulus,  
 $G$  = shear modulus,  $E$  = Young's modulus,  
 $\nu$  = Poisson's ratio,  $A \sim 1$ ,  $E_0$  and  $\rho_0$  pertain to solid.



# Analytic Models



Relationship Between Effective Density and the U-u Hugoniot.





# ***Analytic Models***

- **Stress-Strain Model: Elastic, Perfectly Plastic with Lüders Strain.**

$$\sigma = \sigma_{ys} + C(\varepsilon^P - \varepsilon^L)^n$$

$\sigma$  = effective stress

$\sigma_{ys}$  = initial yield stress

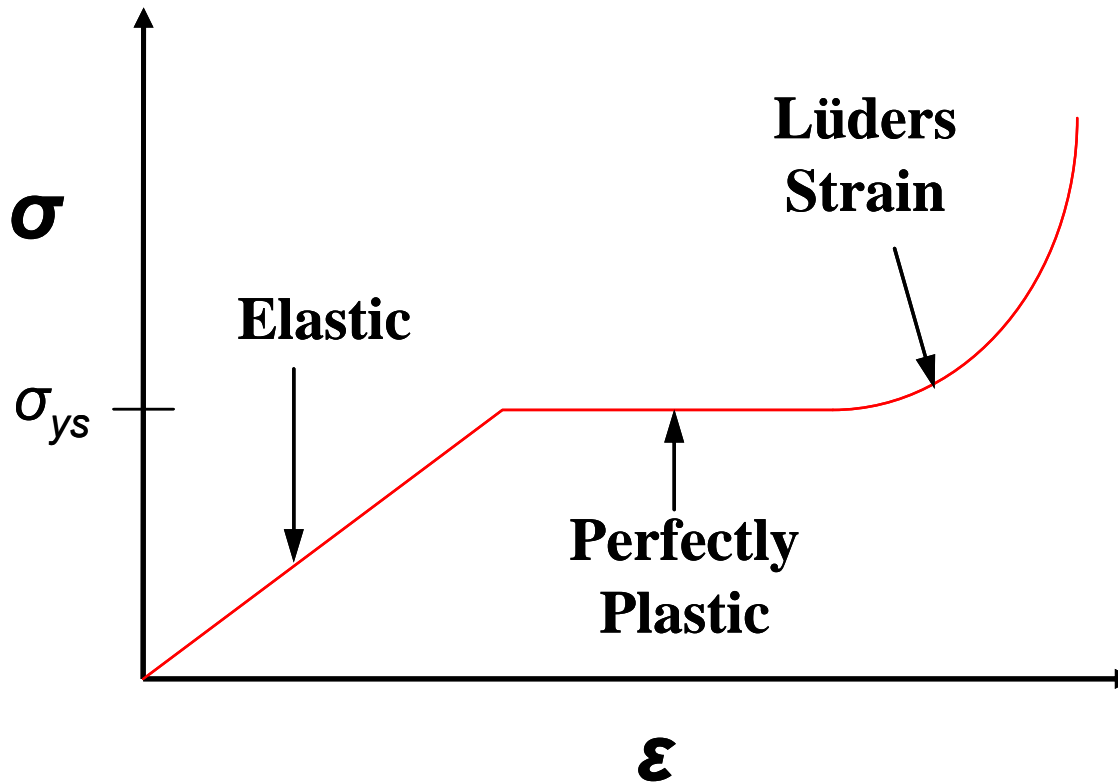
$\varepsilon^P$  = equivalent plastic strain

$\varepsilon^L$  = Lüders strain

$C, n$  = material constants



# *Analytic Models*



Elastic, Perfectly Plastic with Lüders Strain Hardening



# Model Material Properties

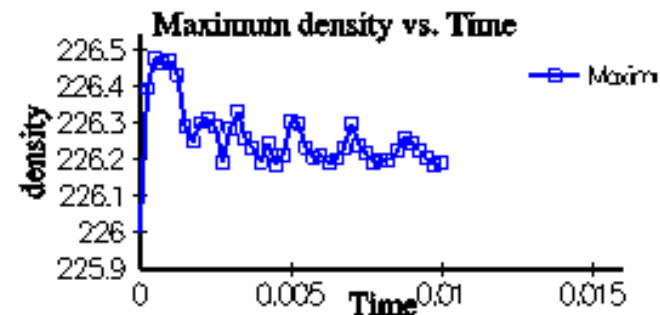
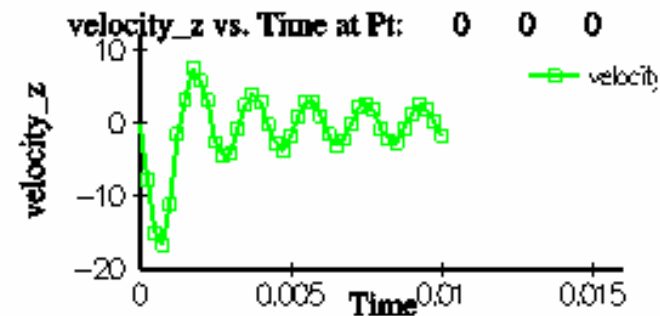
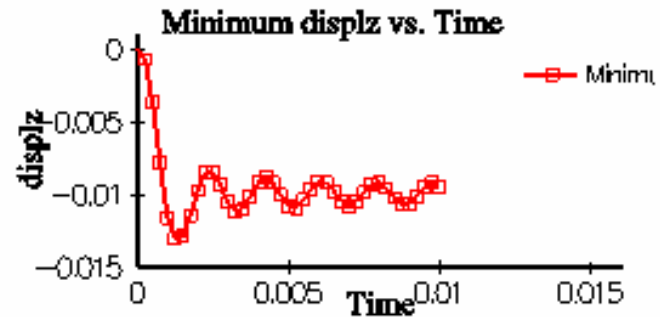
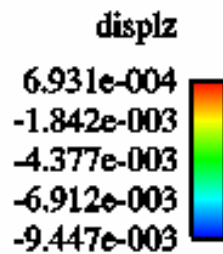
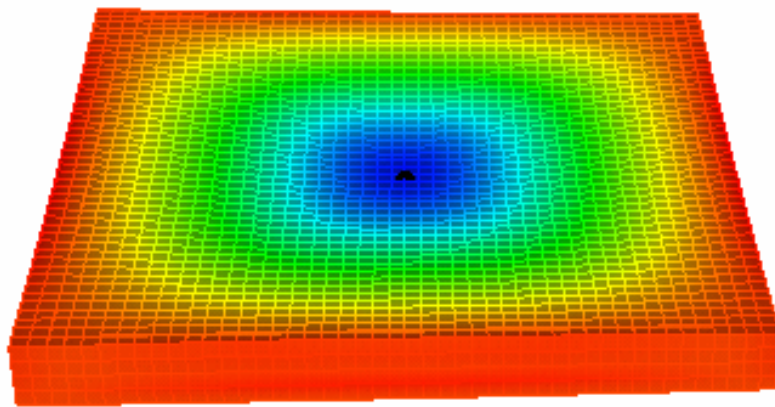
## 40 PPI Aluminum Foam Material Properties.

Property	Value	Source
Young's modulus	452e6 N/m <sup>2</sup>	Reference 2 (data/Gibson-Ashby Equations).
Poisson's ratio	0.33	Reference 6.
Yield stress	1.4e6 N/m <sup>2</sup>	Reference 2 (data/Gibson-Ashby Equations).
Lüders strain	0.002	Reference 8.
Hardening exponent	0.2	Reference 8.
Hardening constant	1.00e09 dyne/cm <sup>2</sup>	Reference 8.
Initial density	226 kg/m <sup>3</sup>	8.1% of value in Reference 5.
C <sub>0</sub>	1,928.3 m/s	Derived from Reference 5 using initial density.
S1	1.338	Reference 5.

2. T. G. Nieh, K. Higashi, and J. Wadsworth, "Effect of Cell Morphology on the Compressive Properties of Open-Cell Aluminum Foams", *Materials Science & Engineering*, A283, 105 – 110, 2000.
4. Paul W. Cooper, "Explosives Engineering", Wiley-VCH, Inc, 1996.
6. Michael F. Ashby et al, "Metal Foams A Design Guide", Butterworth-Heinemann, 2000.
8. Charles M. Stone, Gerald W. Wellman, and Raymond D. Krieg, "A Vectorized Elastic/Plastic Power Law Hardening Material Model Including Luders Strain, Sandia National Laboratories, Sand90-0153, March 1990.



# ALEGRA Simulation of Test 6.





# ***Code Output Comparison***

Comparison of Data vs. ABAQUS, ALEGRA, and CTH.

<b>Parameter</b>	<b>Data</b>	<b>ABAQUS</b>	<b>ALEGRA</b>	<b>CTH</b>
Max. Displacement (m)	0.00961	0.0110	0.0105	0.0075
Max. Foam Velocity (m/s)	Not measured	12.0	7.4	8.75







# ***Conclusion***

- **The shock experiments show that foamed metal can be used to mitigate shocks.**
- **Analyses of the data indicate that because the shocks were mild,  $u_p$  was small, as evidenced by the small change in the calculated density and hand-calculation of the particle velocity via the U-u Hugoniot.**
- **Because the shock was mild, the magnitudes of the compressive and expansive waves were small.**
- **The foam absorbed a small fraction of the shock energy because part of the shock wave encountered air-filled cells (as opposed to aluminum filaments).**





# ***Conclusion***

- **The elastic plastic power law hardening constitutive equation and MG US UP EOS are suitable for modeling metallic foam. The P- $\alpha$  model is not suitable for our study because its usage is limited to void fractions less than 20%.**
- **The ALEGRA, ABAQUS, and CTH results show that the codes were able to calculate the displacement with reasonable success.**

